Influence of Geometrical Imperfections on Local Buckling of GFRP Plates in Civil Engineering

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VARTM: All FRP structures with freedom of shape, size, thickness & layup \odot ... but prone to imperfections \otimes



VARTM production of a beam (Robinson, 2008)

Bio-based FRP bridge in Friesland (2019)

Multidirectional (MD) laminates: Ex = Ey ... Ex >> Ey

TUDelft VARTM = Vacuum Assisted Resin Transfer Moulding (a.k.a. Vacuum Infusion) $_{2}$

Current design verification workflow (EUR 27666EN, CUR96, ...) excludes the influence of imperfections!



Prediction of critical stress is unreliable for plates with out-of-plane geometrical imperfections.



P-w² method: Czapski & Kubiak 2016)

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Imperfections specified for pultruded profiles might not be relevant for VARTM

Pultruded: EN 13706-2

Fabrication tolerances!



VARTM: no standard!

Possible causes of imperfections:

- Moulds
- Curing temperatures
- Shrinkage,
- Large slenderness,
- Various MD laminates

5

GOAL: Build a set of buckling curves based on equivalent geometric imperfections!



Examples: Steel profiles



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Steel plates $\rho = \frac{\overline{\lambda}_p - 0.055 (3 + \psi)}{\overline{\lambda}_p^2} \le 1.0$

 $\rho = \frac{\lambda_p - 0.188}{\overline{\lambda}_p^2} \le 1.0$





3) FRP: Boundary conditions

APPROACH: Non-linear buckling FEA to asses sensitivity of MD laminated plates to imperfections.



CASE: MD laminated plate with simply supported boundary conditions in a range of slenderness



 $t = 4 \div 54 \text{ mm} \longrightarrow \lambda \approx 0.5 \div 4.0$



The buckling resistance is determined as ultimate load in GMNIA





Influence of imperfections is relatively small for slender (thin) plates $\lambda > 1.5!$



Conservative design by: $\sigma_{\rm Ed} \leq \frac{\sigma_{\rm cr} \cdot \eta_{\rm c}}{2}$



γ_M

Stocky (thick) plates with $\lambda < 1.5$ <u>unsafe design</u> by critical stress with significant influence of imperfections!



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CONCLUSION: Prediction of buckling resistance of FRP plates must include imperfections!

- Imperfections of B/125 result in up to 15% additional reduction on stocky plates.
- Resistance of stocky plates is up to 20% overestimated by critical stress approach.
- x2 conservative design of slender plates ($\lambda > 2.5$) based on critical stresses approach.

- Buckling curves approach is feasible to cover aspects of:
 - Different laminates

<- experiment validation

• Different boundary conditions

• Level of (equivalent) geometric imperfections <- measurements UDelft

OUTLOOK: Variation of orthotropy (layup) results in 20% variation of reduction factor



layup III: $E_{cx}/E_{cy} = 1.0$; $D_{11}/D_{22} = 1.45$; $f_{ult} = 182.78 N/mm^2$ layup I: $E_{cx}/E_{cy} = 1.27$; $D_{11}/D_{22} = 1.74$; $f_{ult} = 229.09 N/mm^2$ layup II: $E_{cx}/E_{cy} = 1.80$; $D_{11}/D_{22} = 2.41$; $f_{ult} = 298.59 N/mm^2$

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VALIDATION: The results are independent of aspect ratio, as long as shape and critical stress of the 1st critical mode shape is taken!





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