

Effect of Shear on the Flexure of Pultruded GFRP Tubes

by

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Introductory Remarks

- In **primary structures** circular cross-section **GFRP tubes** are usually used to resist **axial loads**
- Several **experimental studies** have been reported on their **compression buckling** response
- Circular cross-section **GFRP tubes** are rarely used as *beams* in **primary structures**
- **Beams** are required to have **maximum flexural stiffness** with respect to the **major-axis** of flexure
- **Circular** cross-sections have the **same** flexural stiffness for **all** flexural axes
- Hence, **circular** cross-sections are **inefficient** in flexure **compared to** other cross-sectional shapes, e.g. **wide flange (WF) profiles**

Introductory Remarks (Cont'd)

Fig 1: Guard rail comprised of circular cross-section GFRP tubular posts and rails with multi-part mechanically fastened joints



- In secondary structures flexural loads are very much lower than in primary structures
- Pultruded GFRP circular cross-section tubes are widely used in secondary structures such as guardrails/safety barriers and the flexural loads are generally low
- Their advantageous properties: high strength to weight ratio, ease of fabrication, high corrosion resistance, long maintenance-free service life etc. more than outweigh their low flexural stiffness

Introductory Remarks (Cont'd)

- Since **2012** a number of **experimental/numerical** investigations of **GFRP guardrails** fabricated from circular cross-section **tubes** have been carried out in the **Engineering Department**
- The **primary loads** (specified in the codes) are **applied** to the **top rail** normal to the **plane** of the guardrail.
- Hence, both the **top rail** and the **posts** are subjected to **flexure**
- As far as the Authors are aware, the effects of **shear deformation** on the **flexural properties** of the **tubes** has been **ignored**, even though sometimes **span to diameter** ratios may be **low**
- Because the **shear modulus** of **steel** is about **40%** of its **elastic modulus**, shear effects are **ignored**. However, the **corresponding** shear modulus **percentage** for **GFRP** is about **4%**
- Therefore, it is of **interest** to determine how much a **significantly** lower shear modulus **impacts** on the **flexural** response of pultruded **GFRP tubes**
- This issue is **explored** by means of **torsion** and **flexure** testing of the GFRP tubes for a **range of spans**

Constituents of the Pultruded GFRP Tubes

- The **tubes** used in the **experimental** investigations were **manufactured by the pultrusion** process – a long-line process
- The **E-glass fibre reinforcement** is in the following forms:-
 - **Rovings** – bundles of **parallel** fibres which run **longitudinally** along the tube and **provide** its **longitudinal** stiffness and strength
 - **Continuous filament mat (CFM)** which **provides** most of the tubes **circumferential** stiffness and strength
- A **lightweight CFM** is used on the tube's **outer surface** to promote a **resin-rich** surface for **safe** handling and **enhance** corrosion resistance
- The **matrix** which **encapsulates** the fibre reinforcement and **facilitates** load transfer to the fibres is an **isophthalic** polyester resin

Torsion Investigation - Test Specimens

Table 1: Mean values of the geometric properties of the pultruded GFRP tubes used in the torsion tests

Test Number	Mean Torsion Span (L) [mm]	Mean Outer Diameter (D _o) [mm]	Mean Inner Diameter (D _i) [mm]	Mean Cross-Sectional Area (A) [mm ²]	Mean Second Moment of Area (I) [mm ⁴]	Mean Polar Second Moment of Area (J) [mm ⁴]
1	500	50.0	39.8	719	1.84 x 10 ⁵	3.67 x 10 ⁵
2	1000					

- **Two lengths of pultruded GFRP tube** were used in the torsion tests
- The **ends of the tubes** were **cleaned and deburred** prior to **measuring** their dimensions
- **Mean values of their geometric properties** are given in **Table 1** above

Torsion Test – Test Specimens (Cont'd)

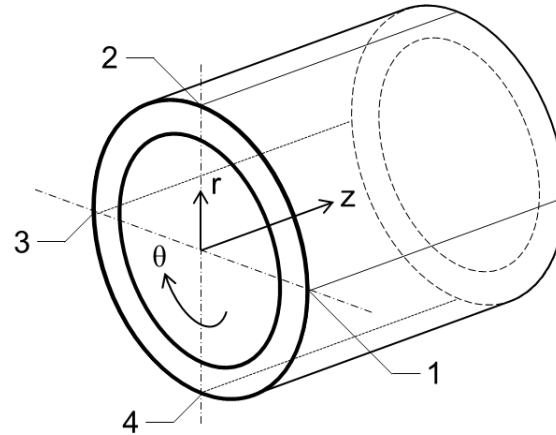


Fig. 2 Cylindrical co-ordinate axes and lines 1 – 4 define the load orientations for the three-point flexure tests

- **Two longitudinal seam lines 180° apart on the tube surface (see Fig. 2) were labelled 1 and 3**
- **A further two lines – 2 and 4 – oriented at 90° to lines 1 and 3 are also shown in Fig. 2. The latter lines were helpful for positioning strain gauges on the tube surface**
- **Vernier calipers were used to measure the inner and outer diameters and wall thicknesses of the tubes at the ends of lines 1 - 4**

Torsion Equations the Determining the In-Plane Shear Modulus

- Assuming the GFRP tubes are **orthotropic with coincident principal material and geometric axes**, then the **outer surface shear stress** is:-

$$\tau_{\theta z} = \tau_{z\theta} = \frac{TD_0}{2J}$$

- The **polar second moment of area** is:-

$$J = \frac{\pi}{32} (D_0^4 - D_i^4)$$

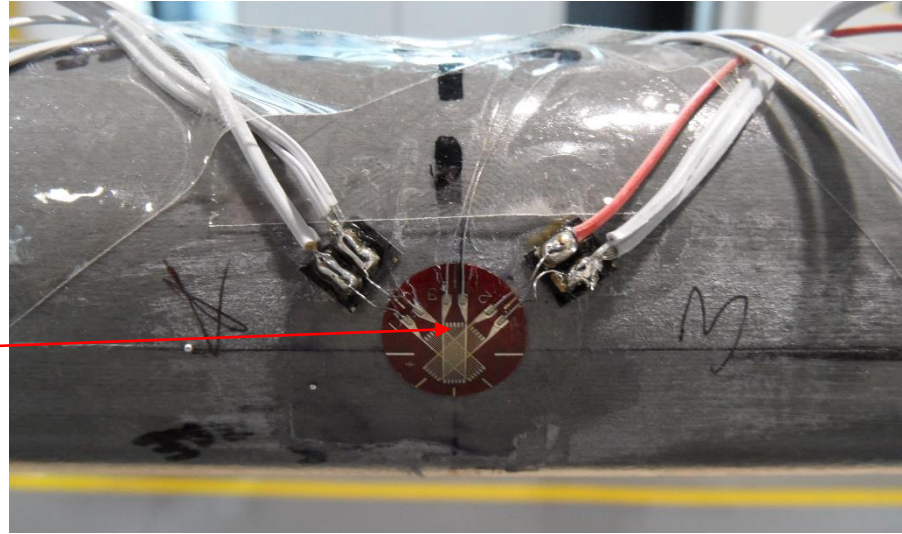
- The **in-plane shear modulus** is:-

$$G_{\theta z} = \frac{\tau_{\theta z}}{\gamma_{\theta z}}$$

- Using the **geometric properties from Table 1** and the **known torque**, the **shear stress** in the tube's surface may be **determined** from the **first two** equations.
- Using the **shear stress** together with the **measured shear strain**, the **shear modulus** may be found from the **third** equation

Tube Instrumentation, Test Set-Up and Procedure

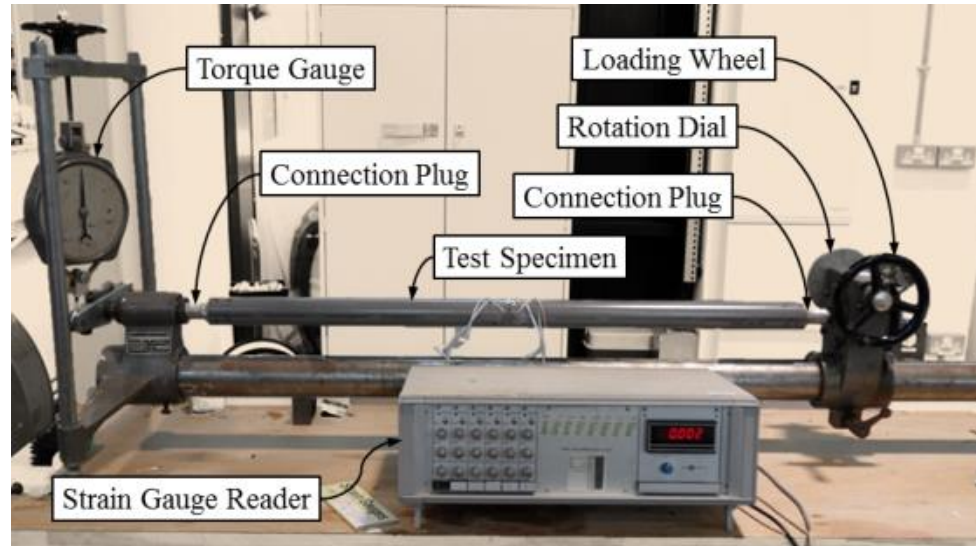
Fig. 3(a): Biaxial strain gauge bonded to the tube wall along Line 1



- The longer tube (1070 mm) had a 35 mm long plug bonded into each end to give a torsion span of 1000 mm
- Two biaxial strain gauges were bonded to the tube at mid-span – one on each of lines 1 and 3
- The gauges had their sensitive axes inclined at $\pm 45^\circ$ to the tube's z-axis
- The tube was installed in the torsion rig and the gauges were connected to the data logger and both strain and torque readings were zeroed

Tube Instrumentation, Test Set-Up and Procedure (Cont'd)

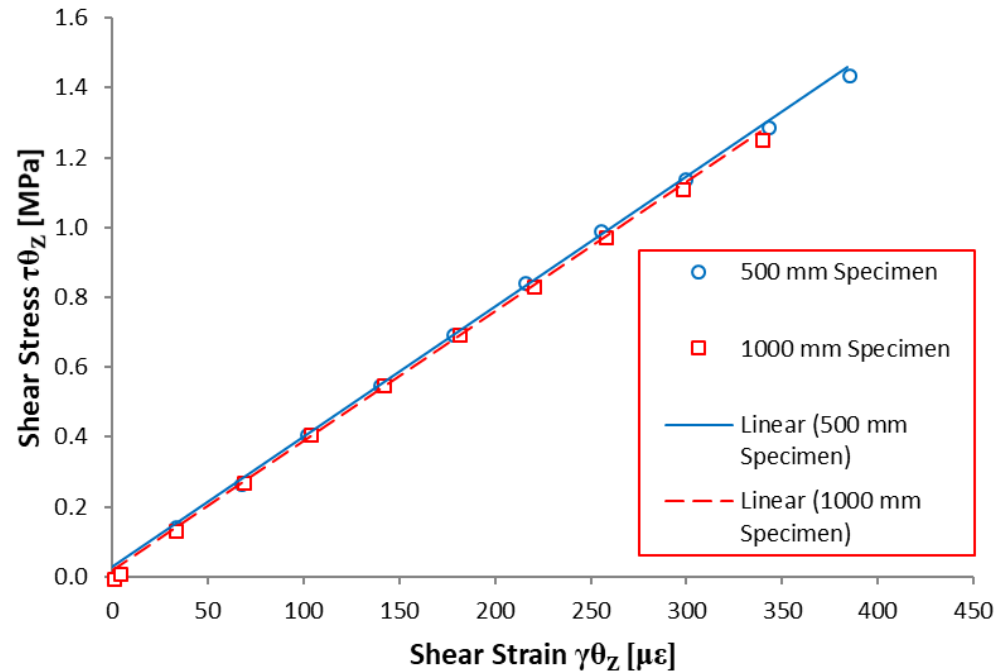
Fig. 3(b): Torsion rig showing the loading arrangement and instrumentation



- The tube was **loaded** in torsion by **axial twist** increments of **1°** using the **loading wheel** up to a **total twist of 10°**
- Readings of **torque** and **strain** were recorded for **each twist increment** during both **loading** and **unloading**. This procedure was **repeated** three times
- The **tube** was then **shortened** to a torsion span of **500 mm** and the **test procedure** was **repeated**

Tube Instrumentation, Test Set-Up and Procedure (Cont'd)

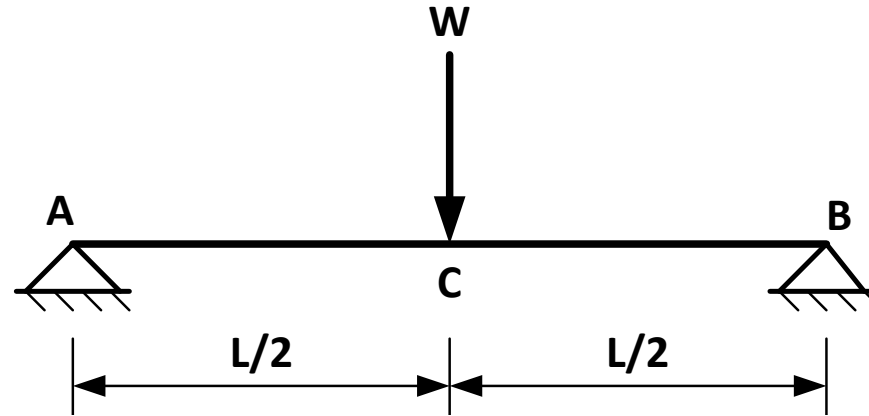
Fig. 4 Shear stress vs shear strain graph of torsion tests on 500 and 1000 mm long pultruded GFRP tubes



- The torsion test data were used to calculate the shear stress for each increment of twist
- The shear strains were determined from the mean biaxial strain data and the shear stress vs shear strain graphs were produced for both spans
- Regression lines were fitted to the data, gave G_{θ_z} values of 3.734 and 3.720 GPa for the 500 and 1000 mm torsion spans, respectively

Three-Point Flexure Investigation – Equations for Elastic Moduli

Fig. 5 Three-point flexural loading on a simply supported pultruded GFRP tube



- Based on **shear-rigid Euler-Bernoulli beam theory**, the tube's **mid-span deflection** is:-

$$\delta_c = \frac{WL^3}{48E_b I}$$

- Likewise, based on **Timoshenko shear deformation theory**, the tube's **mid-span deflection** is:-

$$\delta_c = \frac{WL^3}{48E_{bs} I} + \frac{WL}{4G_{\theta_z} A}$$

- In the above equations E_b is the **elastic bending modulus**, $I = J/2$ is the **second moment of area**, E_{bs} is the **elastic bending-shear modulus** and A is the **cross-sectional area**

Three-Point Flexure Investigation – Equations for Elastic Moduli (Cont'd)

- The deflection equations may be re-arranged to give expressions for the bending and bending-shear moduli:-

$$E_b = \left(\frac{W}{\delta_c} \right) \left\{ \frac{L^3}{48I} \right\} \quad E_{bs} = \frac{E_b}{\left\{ 1 - \frac{L}{4G_{\theta z}A} \left(\frac{W}{\delta_c} \right) \right\}}$$

- Hence, by measuring the mid-span deflection for each three-point flexure test, the quotient $\left(\frac{W}{\delta_c} \right)$ may be determined for each span and substituted into the first equation to give the E_b values
- Likewise, substituting the $\left(\frac{W}{\delta_c} \right)$ and shear modulus $G_{\theta z}$ values into the second equation together with the mean cross-sectional area A for each span into the second equation, the E_{bs} values may be determined
- It is self-evident from the second equation that $E_{bs} \geq E_b$

Three-Point Flexure Test Set-Up and Test Results

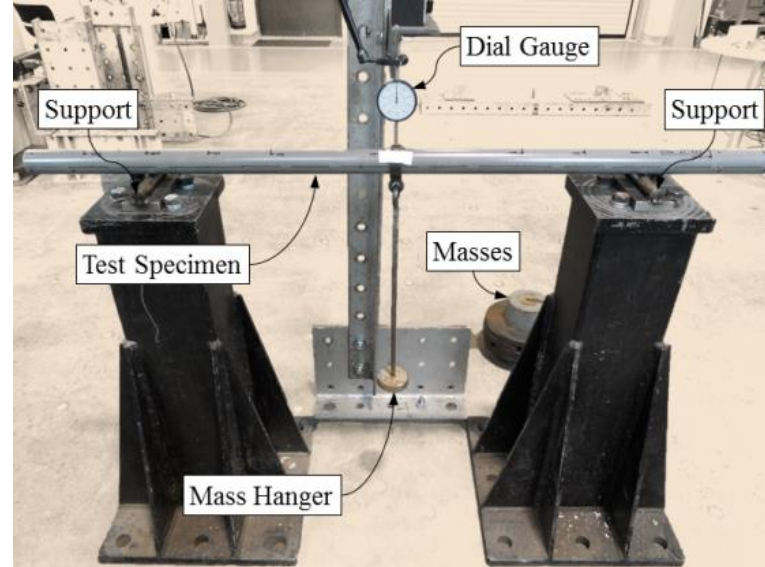


Fig. 6 Experimental test set-up for three-point flexure tests on Pultruded GFRP tubes

- **Three-point flexure** tests were carried out at **spans of 1250, 1000, 750 and 500 mm** on the same tube by **successively** reducing its **original length of 1500 mm**
- For **each span** the tube was **tested three times** with each of the **lines 1 – 4** uppermost in turn
- **Mid-span** deflections were **recorded** for **each** load increment up to a **total load of 981 N**

Three-Point Flexure Test Set-Up and Test Results (Cont'd)

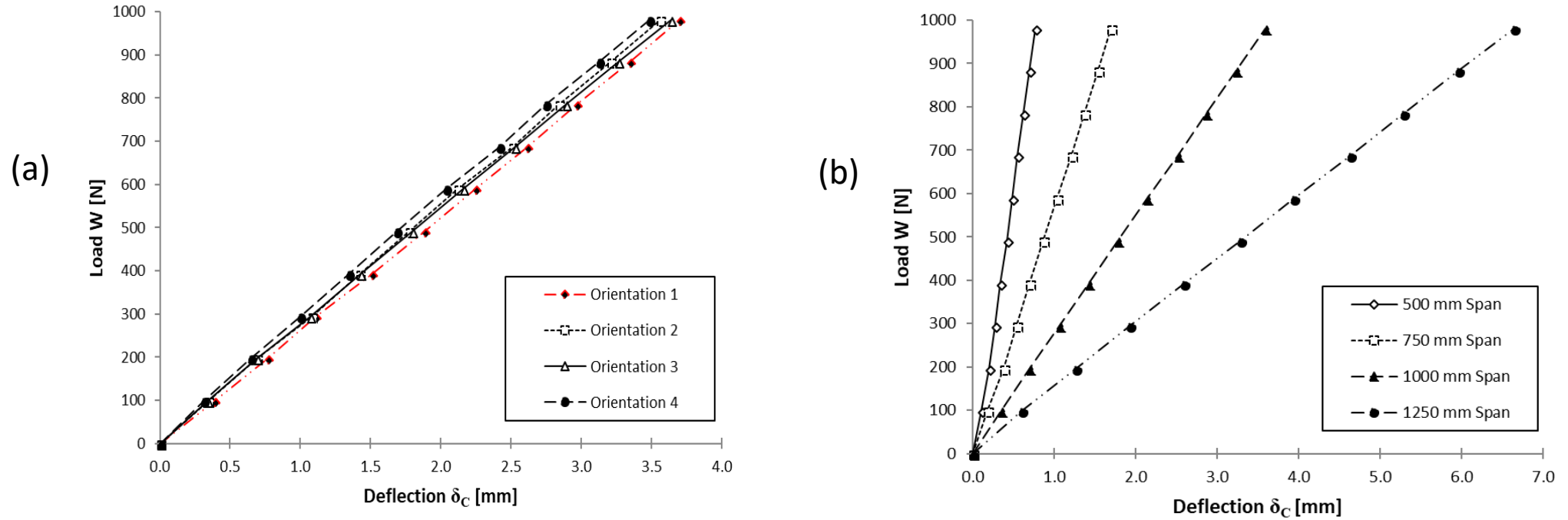


Fig. 7 Load versus mid-span deflection: (a) four orientations of 1000 mm span and (b) mean values of 4 tests at 500, 750, 1000 and 1250 mm spans

- **Fig. 7(a)** shows the load versus mid-span deflection for the four test orientations of the 1000 mm span tube
- **Fig. 7(b)** shows the best-fit straight lines to the twelve load versus mid-span deflection results for each of the four test spans.

Three-Point Flexure Test Set-Up and Test Results (Cont'd)

Table 2 Three-point flexure test spans, elastic moduli and their differences due to shear deformation effects

L [m]	W/δ_c [NM ⁻¹ x 10 ⁵]	E_b [GPa]	E_{bs} [GPa]	$\Delta E = (E_{bs} - E_b)$ [GPa]	$(\Delta E/E_{bs}) \times 100$ [%]
0.50	13.208	18.596	19.808	1.212	6.12
0.75	5.8163	27.655	28.820	1.165	4.04
1.00	2.7220	30.689	31.483	0.794	2.52
1.25	1.4694	32.368	32.929	0.561	1.70

- The mean values of $\frac{W}{\delta_c}$ presented in Table 2 together with the mean value of the in-plane shear modulus $G_{\theta z} = 3.727 \text{ GPa}$ have been used to determine the E_b and E_{bs} values for each span, as shown in Table 2, together with the differences (and percentage differences) due to shear

Concluding Remarks

- The tests on the 1000 mm long pultruded GFRP tube reveal that its flexural stiffness is consistent irrespective of the test orientation
- The same consistency of the flexural stiffness was observed for the other test spans
- The stiffness results consistency observed in the axial torsion and three-point flexure tests suggests that the distribution of the E-glass rovings and CFM is uniform and indicative of the high quality of the manufacturing process – pultrusion
- The test results show that the effect of shear deformation has minimal impact, i.e. less than 6% for the practical range of span to diameter ratios, on the flexural modulus of pultruded GFRP circular cross-section tubes
- Circular cross-section shapes display excellent properties in torsion and resist shear deformation more effectively than flat plates and provide confidence in their use in safety barrier/guardrail applications

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